

FEMCI Workshop

at GSFC

18 May2000

**Nonlinear FEA in the Design of a
High Precision Opto-mechanical Instrument**

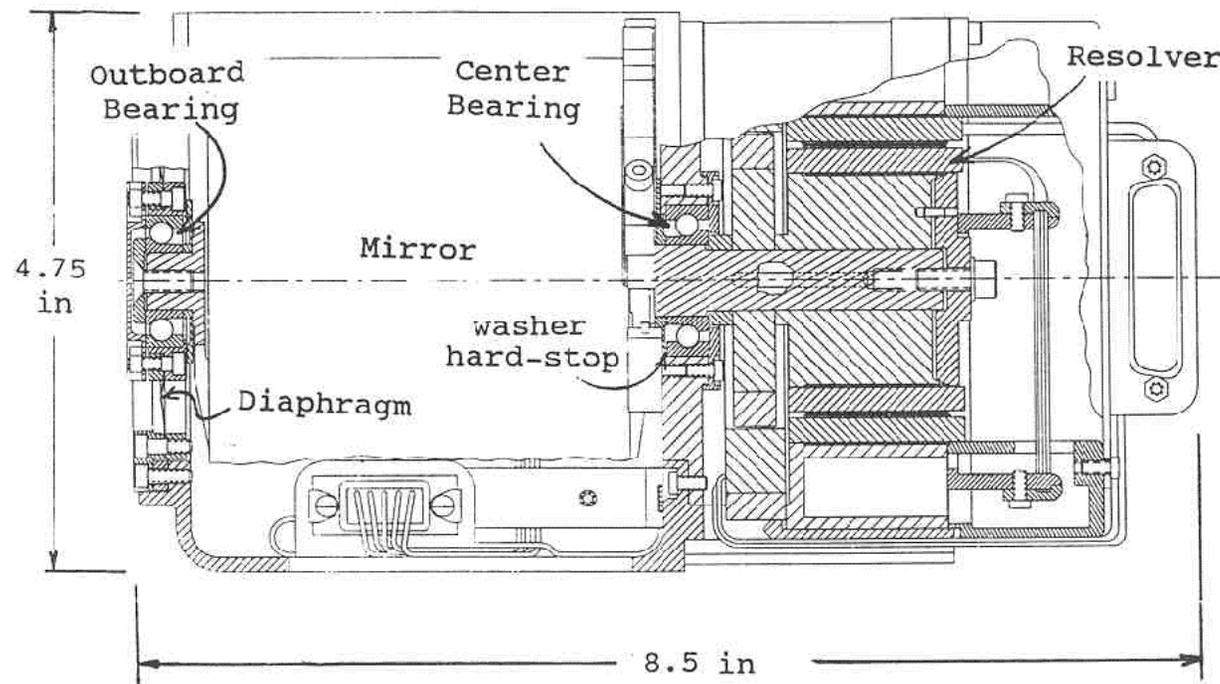
by

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John Traylor - NRL

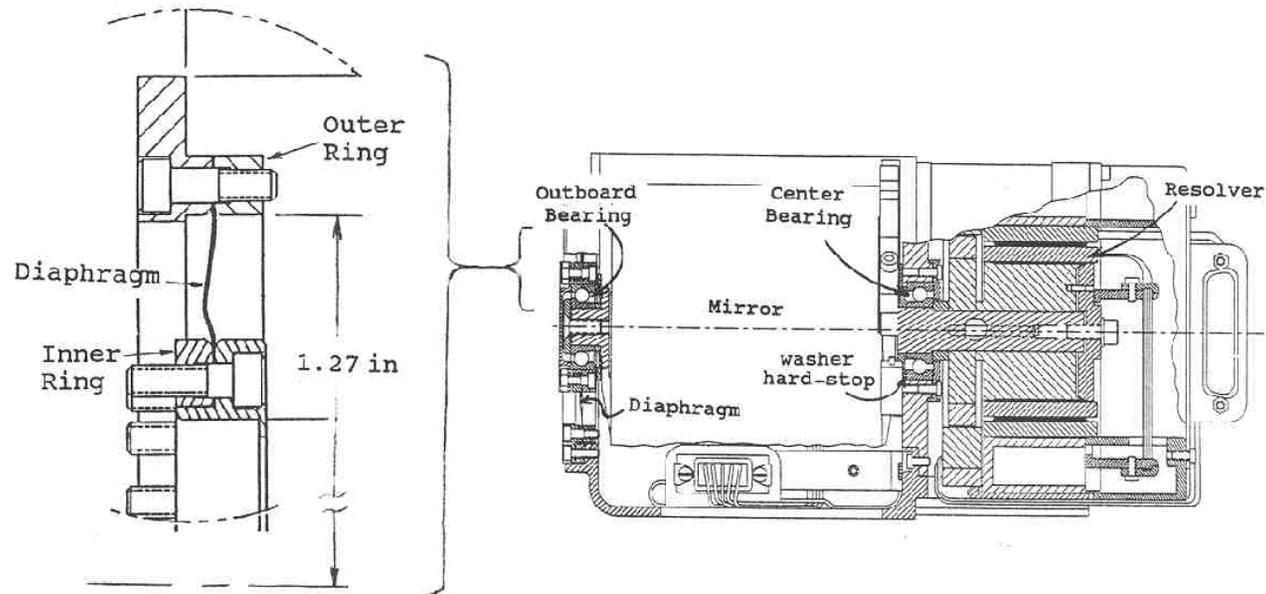
OVERVIEW AND DESCRIPTION

- DESIGN AND ANALYSIS RELATED TO SCAN MIRROR ASSEMBLY (SMA) OF THE “SPECIAL SENSOR ULTRAVIOLET LIMB IMAGER” (SSULI) INSTRUMENT
- WILL FLY ON AN EARTH ORBITING SPACECRAFT TO MEASURE THE DENSITY OF AIR AT ALTITUDES FROM 48KM TO 750KM
- SMA USED TO SWEEP THE FIELD OF VIEW OF A DETECTOR BY ROTATING A SILICON CARBIDE MIRROR, DESIGN LIFE = 3 YEARS



DESIGN ISSUES / DIFFICULTIES

- ANNULAR DIAPHRAGM USED TO APPLY AND MAINTAIN A “LIGHT” AXIAL PRELOAD ON THE BEARING SET
- SURVIVING AXIAL LAUNCH LOADS DESPITE LIGHT PRELOAD AND FLEXIBLE DIAPHRAGM
- TRAVEL STOP INCORPORATED TO LIMIT LAUNCH VIBRATION LOADS
- AMOUNT OF FREE TRAVEL: A COMPROMISE BETWEEN HIGHER MECHANICAL LOADS DURING LAUNCH AND AN INCREASED RISK OF CONTACT DURING OPERATION



- PRELIMINARY SIZING OF DIAPHRAGM, BEARINGS, INTERFACES
- DETAILED ANALYSES: DIAPHRAGM STIFFNESS & STRUCTURAL INTEGRITY
DIAPHRAGM SUPPORTED SYSTEM DYNAMICS
- VERIFICATION: STRUCTURAL TESTING

APPLIED LOADING

- ESTIMATE OF STATIC ACCELERATION DUE TO RANDOM VIBRATION LOADS:

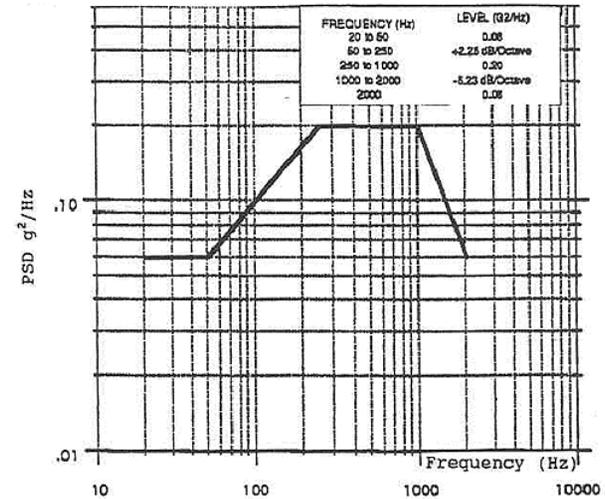
$$a_{3s} = 3 \left(\Pi f_1 PSD \frac{Q}{2} \right)^{\frac{1}{2}} = 3 \left(\Pi (170) (0.14) \left(\frac{10}{2} \right) \right)^{\frac{1}{2}} = 58G' s$$

- LAUNCH QUASI-STATIC LOADS:
TITAN-III, THRUST: 6 G's, LATERAL: 2.6 G's

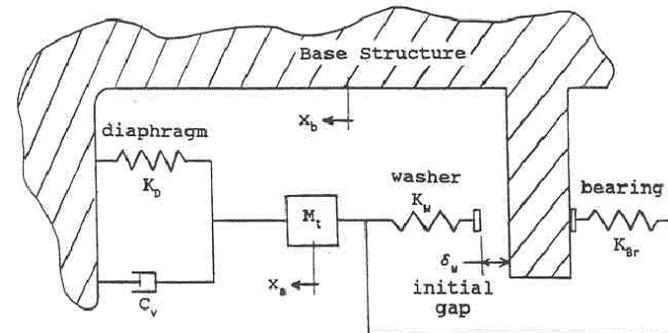
$$a_{QS} = \left(6^2 + 2.6^2 \right)^{\frac{1}{2}} = 6.5G' s$$

- SIMPLIFIED WORST CASE LOADING:
APPLIED PRELOAD (6.5 lb) + ENFORCED DISPLACEMENT (0.010 in)

RANDOM VIBRATION LOADS:

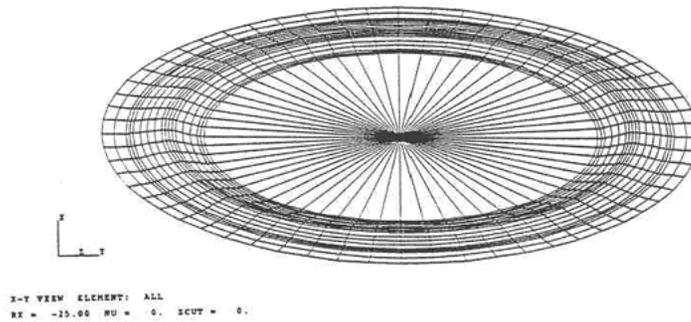


MECHANICAL SYSTEM SCHEMATIC:

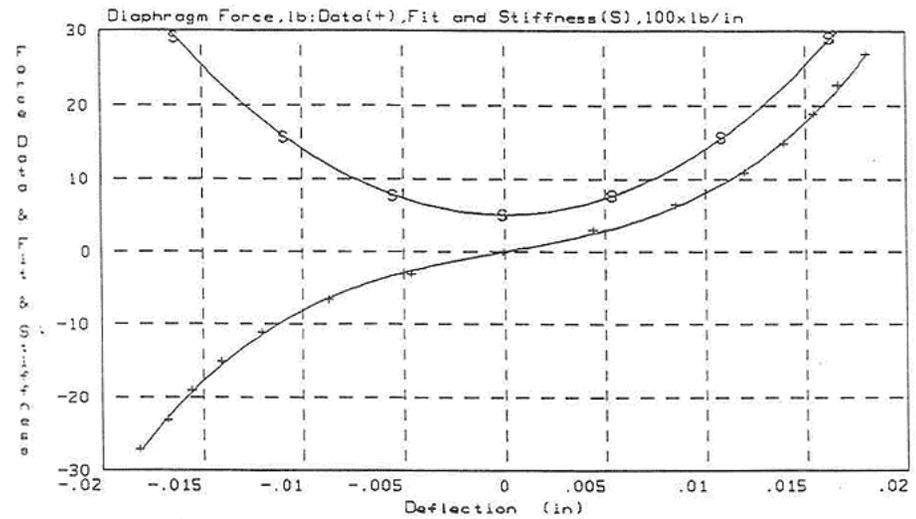


FORCE - DEFLECTION RESPONSE

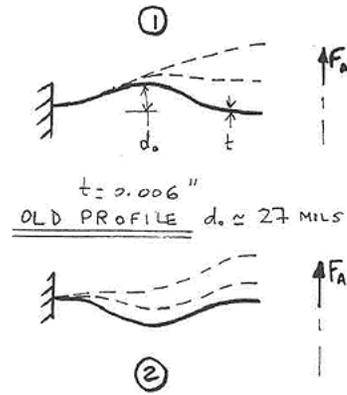
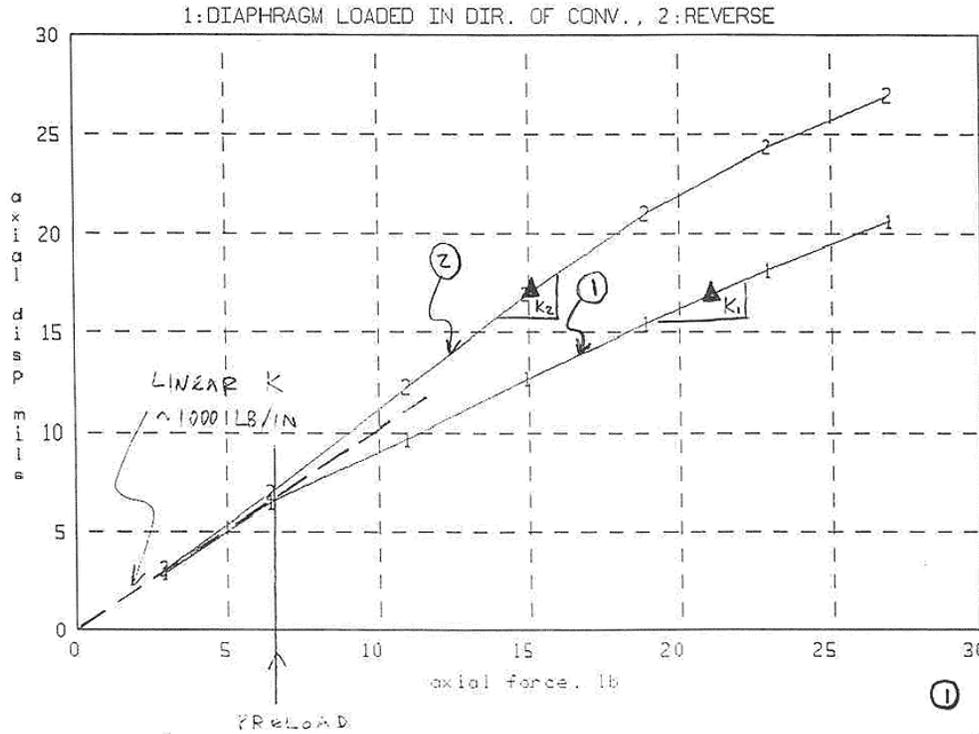
- NONLINEAR FEA OF DIAPHRAGM USING NASTRAN



- FORCE-DISPLACEMENT RESPONSE



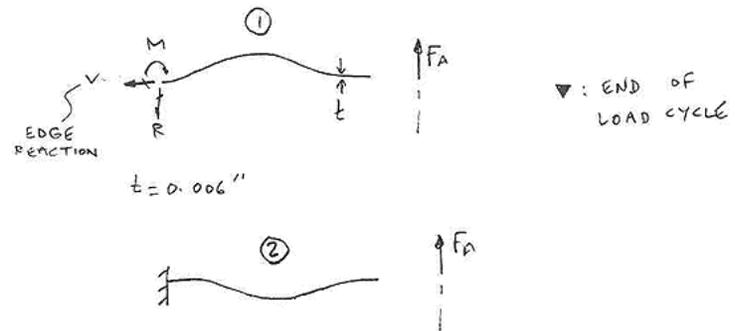
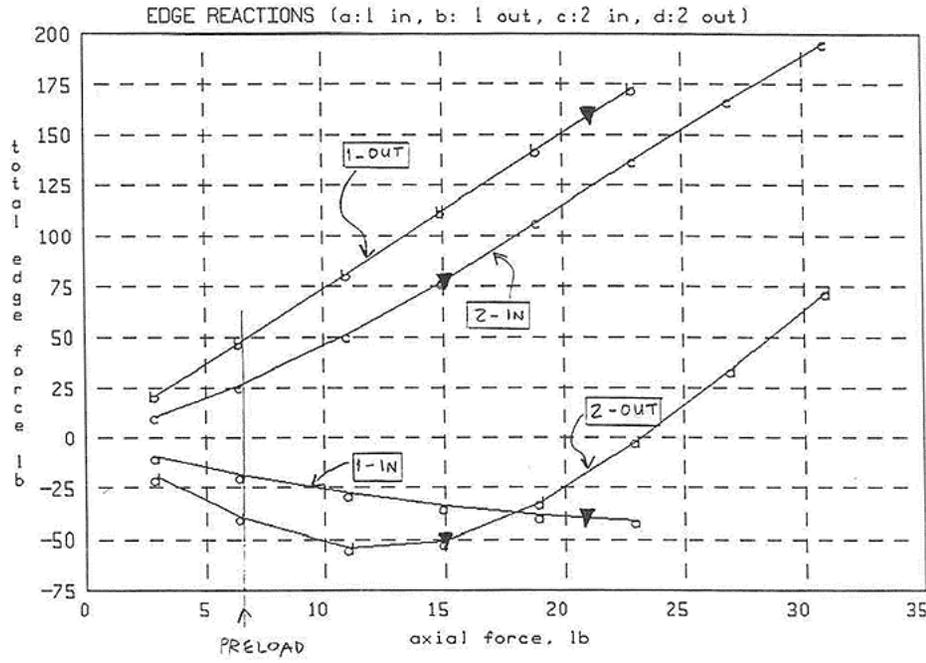
- NONLINEARITY DUE TO FLATTENING OF THE DIAPHRAGM CONVOLUTION
- TEST SHOWED 1000 lb/in AT APPLIED FORCE = 7lb
- ANALYTICAL SOLUTION GIVES 540 lb/in AT 0 APPLIED FORCE



▼ : END OF LOAD
CYCLE
(6.5 LB PRELOAD)
+ 0.01" DISP

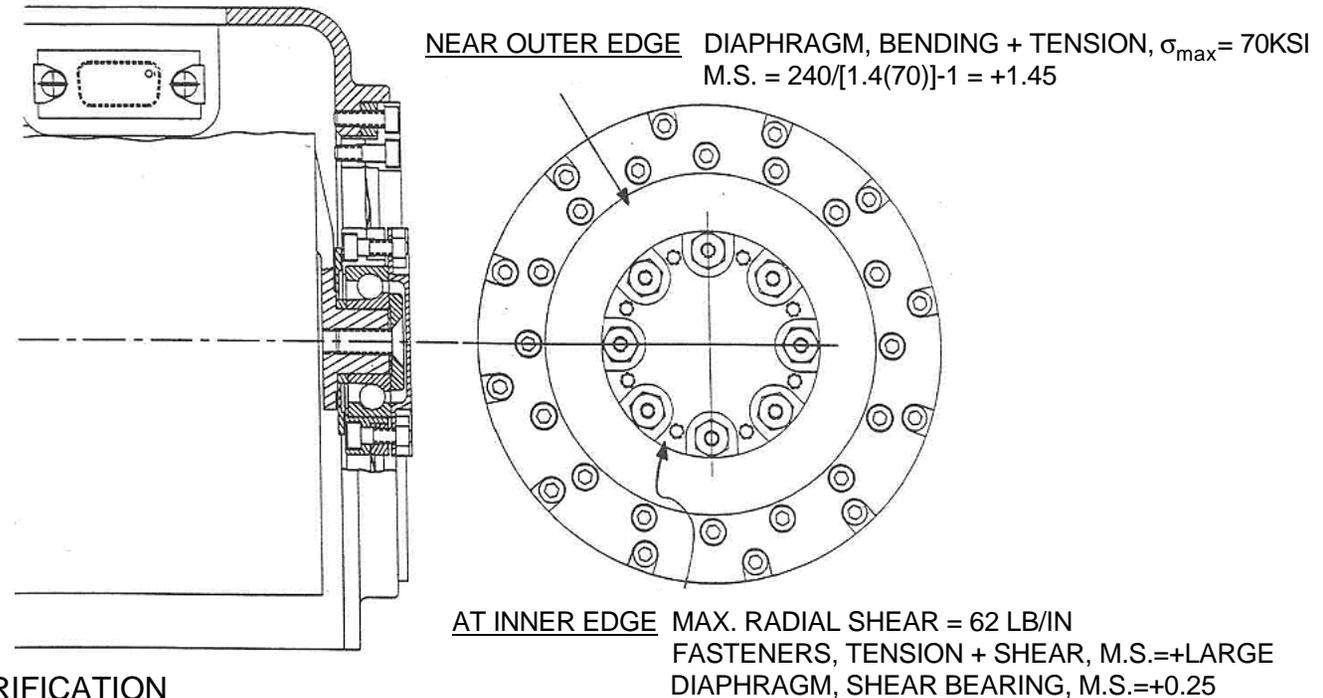
$K_2 \approx 890$ LB/IN

$K_1 \approx 1430$ LB/IN



CONVOLUTED DIAPHRAGM STRUCTURAL INTEGRITY

- DIAPHRAGM AND INTERFACES STRENGTH ASSESSMENT:

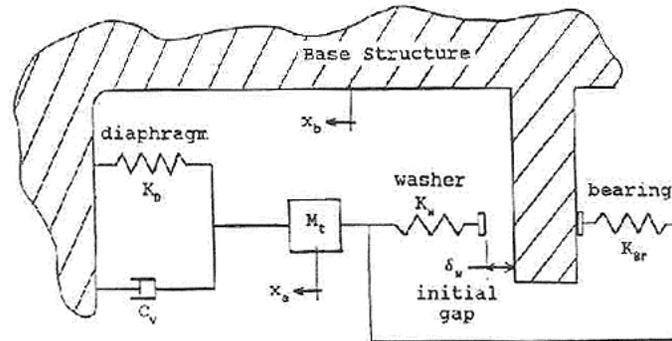


- CYCLIC LIFE VERIFICATION

- A THROUGH-THE-THICKNESS CRACK WAS ASSUMED
- IT WAS PROPAGATED IN THE CIRCUMFERENTIAL DIRECTION
- PROPAGATED UNDER FULLY REVERSED CYCLING AT THE MAX. STRESS LEVEL
- USING THE FRACTURE ANALYSIS ALGORITHM "FLAGRO"
- BASED ON A SCATTER FACTOR OF 4, A LIFE OF 18,400 CYCLES WAS ESTIMATED
- ACTUAL NO. OF CYCLES THE PART WILL SEE ESTIMATED TO BE 11,800 CYCLES

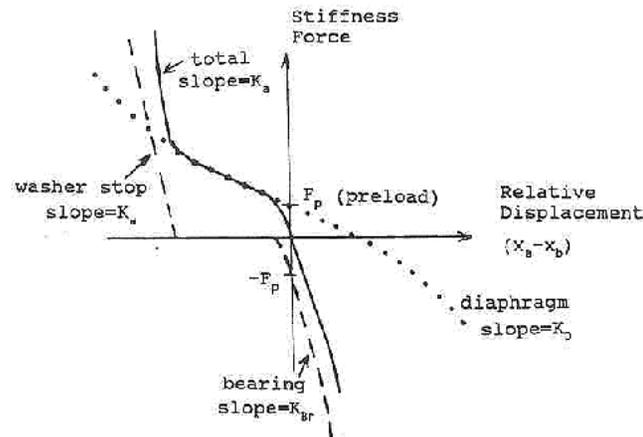
AXIAL DYNAMIC RESPONSE

- MECHANICAL SYSTEM SCHEMATIC:



- EQN. OF MOTION: $M_t d^2 x_a / dt + C_v dx_a / dt + K_a x_a = G_a M_t + K_a x_b$

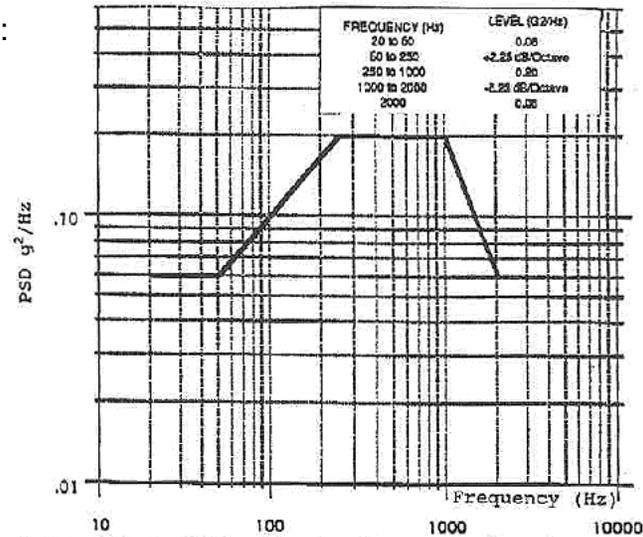
- SYSTEM STIFFNESS FORCE - DEFLECTION:



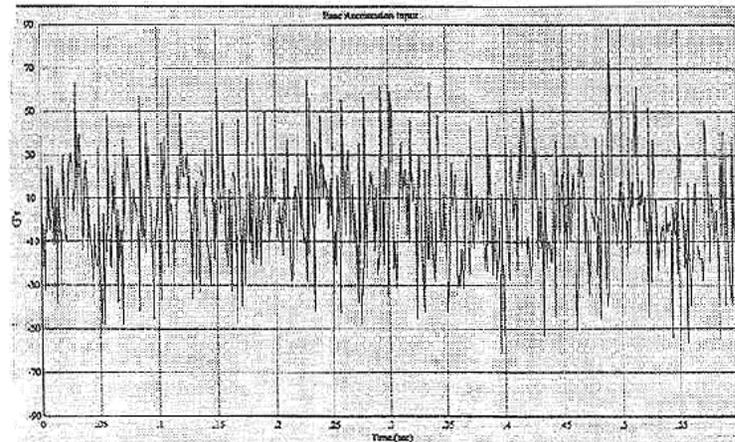
AXIAL RESPONSE SIMULATION

- PSD OF TEST RANDOM BASE SHAKE:

QUALIFICATION LEVEL
RANDOM VIBRATION
TEST PSD

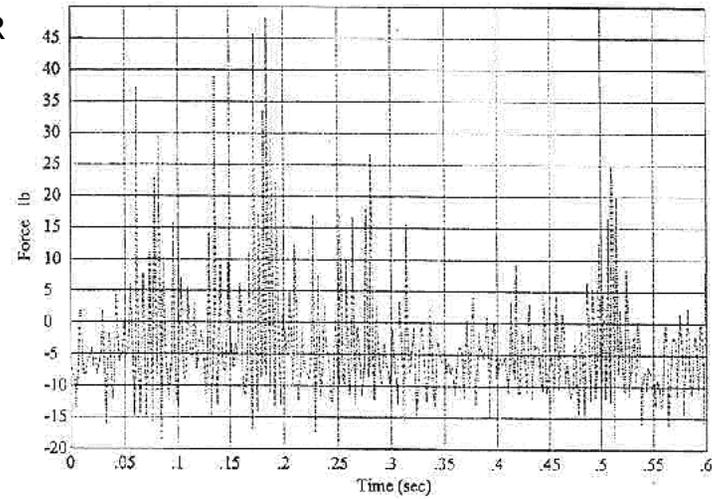


- BASE ACCELERATION INPUT:
(G's vs. TIME)

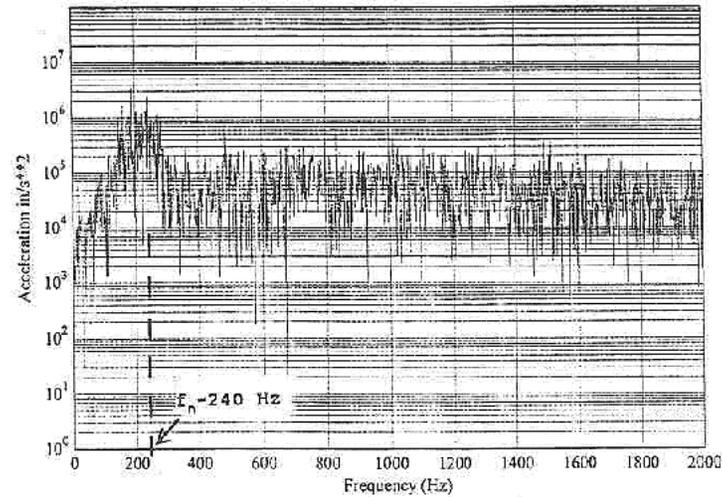


AXIAL RESPONSE SIMULATION

- SIMULATION RESULTS FOR DIAPHRAGM FORCE:



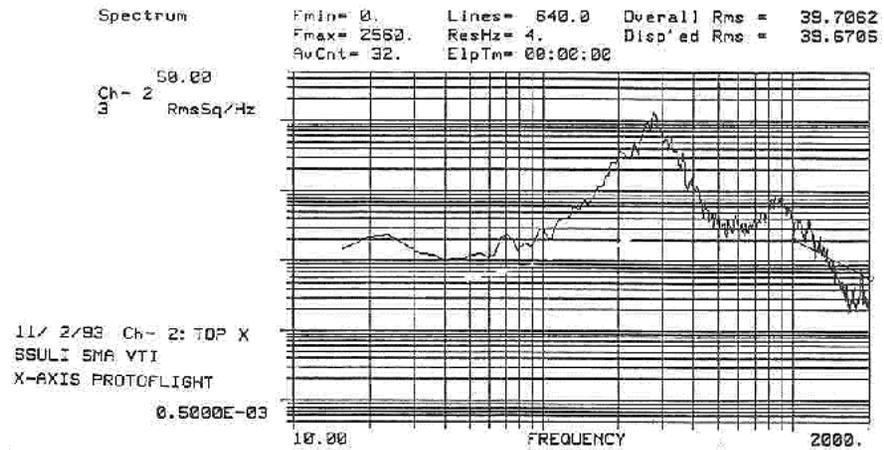
- ACCELERATION RESPONSE FREQUENCY CONTENT:



STRUCTURAL VERIFICATION TESTING

- RANDOM VIBRATION TESTS ON SMA MASS MODEL
- IN THE AXIAL AND THE TWO RADIAL DIRECTIONS

AXIAL ACCELERATION UNDER
AXIAL BASE SHAKE:



RADIAL ACCELERATION UNDER
RADIAL BASE SHAKE:

